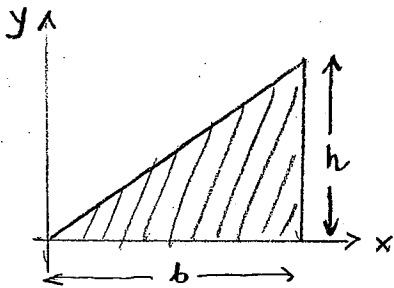


EM311M - Dynamics

Exam 3

Monday, Dec 1, 2008, 6:00-8:00 p.m., WEL 3.502

1. Use Cartesian coordinates to compute the product of inertia I_{xy} , for a homogeneous triangular plate of mass m shown below. (5 points)



$$\begin{cases} 0 < x < b \\ 0 < y < \frac{h}{b}x \end{cases} \quad m = \frac{1}{2}bh\rho \Rightarrow \rho = \frac{2m}{bh}$$

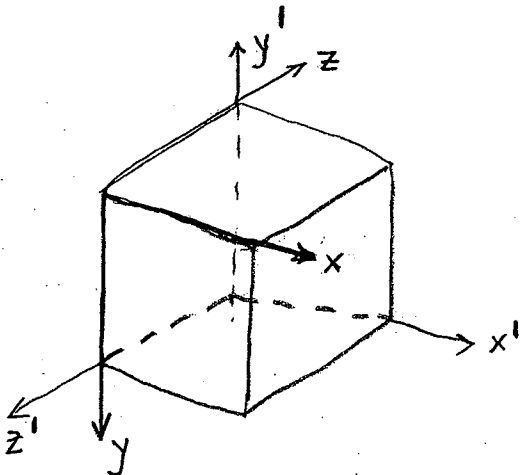
$$I_{xy} = \rho \int_0^b \int_0^{\frac{h}{b}x} xy \, dy \, dx = \rho \int_0^b x \left[\frac{y^2}{2} \right]_0^{\frac{h}{b}x} dx$$

$$= \frac{\rho}{2} \int_0^b x \frac{h^2}{b^2} x^2 dx = \frac{\rho h^2}{2b^2} \left[\frac{x^4}{4} \right]_0^b = \frac{\rho}{2} \frac{h^2}{b^2} \frac{b^4}{4}$$

$$= \frac{\rho}{8} b^2 h^2 = \frac{2m}{bh \cdot 8} b^2 h^2 = \frac{1}{4} m b h \quad [\text{kg}\cdot\text{m}^2]$$

(5)

2. Define the transformation matrix from system $Oxyz$ to system $O'x'y'z'$ and compute it for the systems shown below. (5 points)



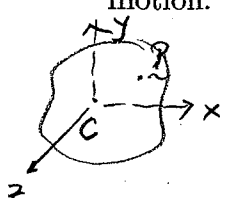
$$\alpha_{ij} = \underline{e}'_i \cdot \underline{e}_j \quad (1)$$

$$\underline{a} = \begin{pmatrix} 1 & 0 & 0 \\ 0 & -1 & 0 \\ 0 & 0 & -1 \end{pmatrix} \begin{matrix} \underline{e}'_1 \text{ in } xyz \\ \underline{e}'_2 \text{ in } xyz \\ \underline{e}'_3 \text{ in } xyz \end{matrix}$$

$$\begin{matrix} \underline{e}'_1 \text{ in } x'y'z' \\ \underline{e}'_2 \text{ in } x'y'z' \\ \underline{e}'_3 \text{ in } x'y'z' \end{matrix}$$

(4)

3. Derive the formula for the kinetic energy of a rigid body undergoing an arbitrary planar motion. (5 points)



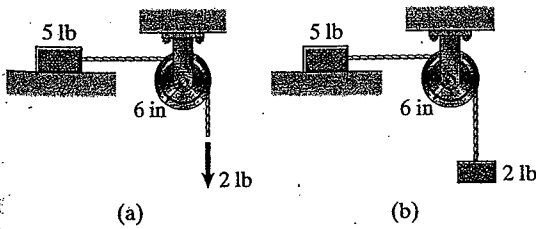
$$\underline{v}_P = \underline{v}_C + \underline{\omega} \times \underline{r}_{CP} = (v_{Cx}, v_{Cy}, 0) + \begin{matrix} \omega(0, 0, \omega) \\ \underline{r}_{CP}(x, y, 0) \\ (-\omega y, \omega x, 0) \end{matrix}$$

$$= (v_{Cx} - \omega y, v_{Cy} + \omega x, 0)$$

$$\int \rho (v_{Cx} - \omega y)^2 + (v_{Cy} + \omega x)^2 = (v_{Cx}^2 + v_{Cy}^2) \int \rho - 2\omega v_{Cx} \int \rho y + 2\omega v_{Cy} \int \rho x$$

$$T = \frac{1}{2} \int \rho \underline{v}_P^2 = \frac{1}{2} m \underline{v}_C^2 + \frac{1}{2} I_C \omega^2 + \underbrace{\omega^2 \int \rho (y^2 + x^2)}_{I_C} \quad (5)$$

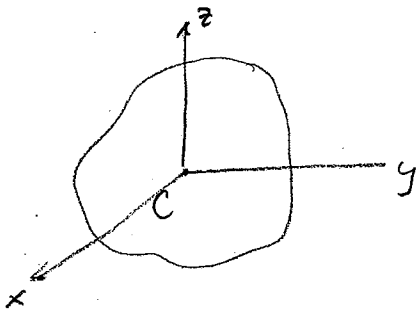
4. Review the two scenarios below. In which case do you expect the acceleration of 5 lb block be smaller? Explain. (5 points)



In case (b). Forces are the same, but the inertia is bigger.

(5)

5. Derive the equation of angular motion for a rigid body undergoing an arbitrary motion, in the body-fitted system of coordinates (5 points)



x, y, z - a body-fitted system with center at \underline{c} .

Principle of angular impulse and momentum

$$\left(\frac{I}{\underline{c}} \underline{\omega} \right)' = \underline{M}_{\underline{c}}$$

In a rotating system: $\dot{\underline{f}} = \dot{\underline{f}}/rel + \underline{\Omega} \times \underline{f}$

In a body-fitted system: $\underline{\Omega} = \underline{\omega}$

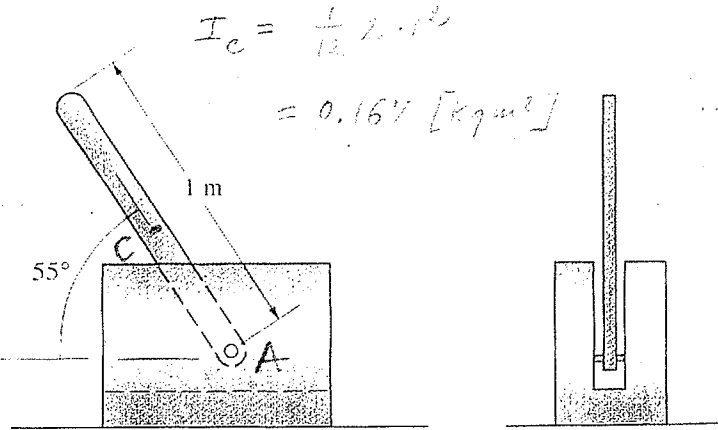
Hence $\left(\frac{I}{\underline{c}} \underline{\omega} \right)' / rel + \underline{\omega} \times \left(\frac{I}{\underline{c}} \underline{\omega} \right) = \underline{M}_{\underline{c}}$

$$\frac{I}{\underline{c}} \dot{\underline{\omega}} / rel$$

(5)

5

6. The 2-kg slender bar and k-kg block are released from rest in the position shown. If friction is negligible, what is the block's acceleration at that instant? (25 points)

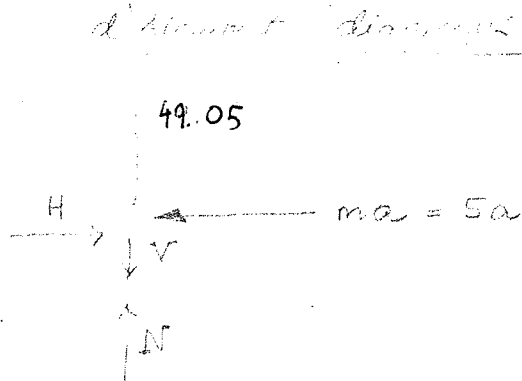
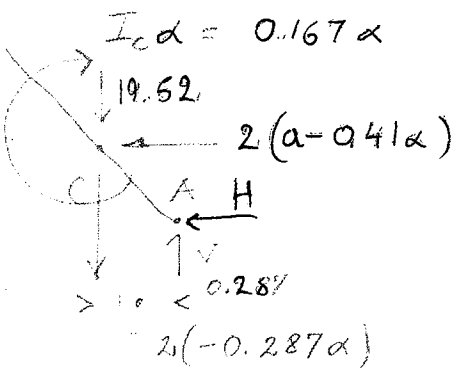


$$I_c = \frac{1}{12} 2 \cdot 1^2 = 0.167 \text{ [kgm}^2]$$

$$\underline{a}_c = \underline{a}_A + \underline{\alpha} \times \underline{r}_{AC} - \omega^2 \underline{r}_{AC}$$

$$= (a, 0, 0) + \alpha (0, 0, \alpha) \times (-0.287, 0.410, 0)$$

$$= (-0.41\alpha, -0.287\alpha, 0)$$



Bar: $\sum \tau_c = 0$

$$19.62 \cdot 0.287 + 2(-0.287\alpha) \cdot 0.287 + 2(a - 0.41\alpha) \cdot 0.41 - 0.167a = 0$$

$$\Rightarrow 0.821a = -5.631 + 0.165\alpha + 0.336\alpha + 0.167\alpha$$

$$= -5.631 + 0.668\alpha \quad (1)$$

Bar + block: $\sum F_x = 0$

$$-2(a - 0.41\alpha) - 5a = 0$$

$$7a = 0.82\alpha \Rightarrow a = 0.117\alpha \quad (2)$$

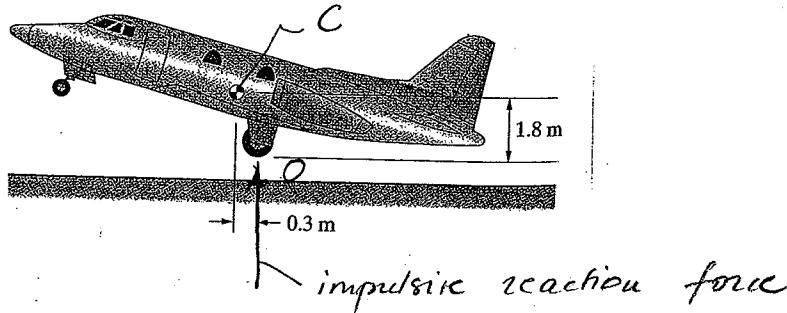
Substituting (2) into (1):

$$0.096\alpha = -5.631 + 0.668\alpha$$

$$\therefore \alpha = 9.844 \text{ [rad/s}^2]$$

$$\Rightarrow a = 1.151 \text{ [m/s}^2]$$

7. The horizontal velocity of the landing airplane (its center of mass) is 50 m/s, its vertical velocity is 2 m/s, and its angular velocity is zero. The mass of the airplane is 12,000 kg, and the moment of inertia about its center of mass is 100,000 kg-m². The coefficient of restitution of the landing impact is $e = 0.4$. Determine the angular velocity of the airplane after the impact. (25 points)



- Principle of angular impulse and momentum wrt point O on the ground:

Eq. 1
$$\vec{OC} \times m \vec{v}_C^{\text{before}} + (0, 0, I_C \omega_C^{\text{before}}) = \vec{OC} \times m \vec{v}_C^{\text{after}} + (0, 0, I_C \omega_C^{\text{after}})$$

$$\begin{array}{r} \times \vec{OC} (-0.3, 1.8, 0) \\ m \vec{v}_C^{\text{before}} (-50, -2, 0) \\ \hline (0, 0, 90.6) \end{array}$$

- Principle of linear momentum in x-direction

$$m v_{Cx}^{\text{before}} = m v_{Cx}^{\text{after}} \Rightarrow v_{Cx}^{\text{after}} = v_{Cx}^{\text{before}} = -50 \quad (5)$$

- Coefficient of restitution $v_{Oy}^{\text{after}} = -0.4 v_{Oy}^{\text{before}} = 0.8 \quad (5)$

- Kinematics
$$\vec{v}_C^{\text{after}} = \vec{v}_O^{\text{after}} + \omega^{\text{after}} \times \vec{OC}$$

$$= (v_{Ox}^{\text{after}}, 0.8, 0) + \omega^{\text{after}} \times \vec{OC} (-0.3, 1.8, 0)$$

$$= (v_{Ox}^{\text{after}}, 0.8, 0) + \omega^{\text{after}} (-1.8, -0.3, 0)$$

Hence
$$v_{Oy}^{\text{after}} = 0.8 - 0.3 \omega^{\text{after}}$$

(5)

• angular momentum wrt point O after impact

$$\begin{aligned} & \times \vec{OC} (-0.3, 1.8, 0) \\ & m \vec{v}_c^{after} (-50m, (0.8 - 0.3\omega^{after})m, 0) \quad + \quad (0, 0, I_c \omega^{after}) \end{aligned}$$

$$(0, 0, -0.3(0.8 - 0.3\omega^{after})m + 90m)$$

Final equation for ω : (from Eq. 1)

(10)

$$90.6m = -0.3(0.8 - 0.3\omega^{after})m + 90m + I_c \omega^{after}$$

Divide by m

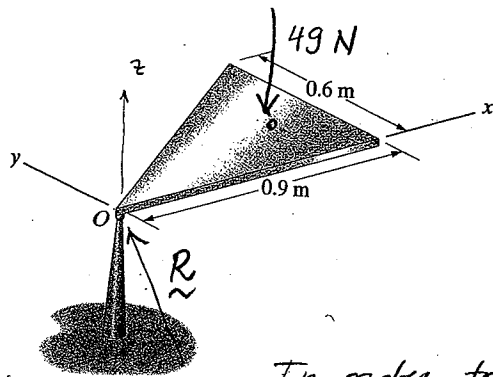
$$90.6 = -0.3(0.8 - 0.3\omega^{after}) + 90 + \frac{I_c}{m} \omega^{after}$$

8.33

$$(8.33 + 0.09) \omega^{after} = 0.6 + 0.24 = 0.84$$

$$\omega^{after} \approx 0.1 \left[\frac{\text{rad}}{\text{s}} \right]$$

8. If the 5-kg plate is released from rest in the horizontal position, what force is exerted on it by the ball-and-socket support at that instant? (25 points)



Eqn's of translational motion:

$$\begin{aligned} 5a_{cx} &= R_x \\ 5a_{cy} &= R_y \\ 5a_{cz} &= R_z - 49 \end{aligned} \quad (1) \quad (5)$$

In order to determine \vec{R} , we need to compute \vec{a}_c

We use the body-fitted system $x'y'z'$

$$\begin{aligned} \vec{I}_O \vec{\dot{\omega}} + \vec{\omega} \times \vec{I}_O \vec{\omega} &= \vec{M}_O = \vec{OC} (0.6, 0.2, 0) \\ &= \vec{W} (0, 0, -49) \\ &= (-9.81, 29.4, 0) \end{aligned} \quad (5)$$

Inertia: Use parametrization from Problem #1

$$\begin{aligned} I_x &= \rho \int_0^b \int_0^{\frac{h}{b}x} y^2 dy dx = \rho \int_0^b \frac{y^3}{3} \Big|_0^{\frac{h}{b}x} dx = \frac{\rho}{3} \frac{h^3}{b^3} \int_0^b x^3 dx \\ &= \frac{\rho}{3} \frac{h^3}{b^3} \frac{b^4}{4} = \frac{\rho}{12} b h^3 = \frac{2}{12} \rho \frac{bh}{2} h^2 = \frac{1}{6} m h^2 = 0.3 \text{ kgm}^2 \end{aligned}$$

$$\begin{aligned} I_y &= \rho \int_0^b \int_0^{\frac{h}{b}x} x^2 dy dx = \rho \int_0^b x^2 y \Big|_0^{\frac{h}{b}x} dx = \rho \frac{h}{b} \frac{x^4}{4} \Big|_0^b \\ &= \frac{\rho}{4} h b^3 = \frac{1}{2} m b^2 = 2.025 \text{ kgm}^2 \end{aligned}$$

$$I_{xy} = \frac{1}{4} m b h = 0.675 \text{ kgm}^2$$

$$\begin{pmatrix} 0.3 & -0.675 & 0 \\ -0.675 & 2.025 & 0 \\ 0 & 0 & 2.325 \end{pmatrix} \begin{pmatrix} \alpha_x \\ \alpha_y \\ \alpha_z \end{pmatrix} = \begin{pmatrix} -9.81 \\ 29.4 \\ 0 \end{pmatrix} \quad (10)$$

$$\begin{aligned} \alpha_z &= 0 \\ \alpha_x &= \frac{\begin{vmatrix} -9.81 & -0.675 \\ 29.4 & 2.025 \end{vmatrix}}{\begin{vmatrix} 0.3 & -0.675 \\ -0.675 & 2.025 \end{vmatrix}} = -\frac{0.02}{0.15} \approx -0.133 \text{ [rad/s}^2\text{]} \\ \alpha_y &= \frac{\begin{vmatrix} 0.3 & -9.81 \\ -0.675 & 29.4 \end{vmatrix}}{0.15} \approx 14.5 \text{ [rad/s}^2\text{]} \end{aligned}$$

Kinematics:

$$\begin{aligned} \underline{a}_G &= \cancel{\underline{a}_O} + \underline{\omega} \times \underline{OG} + \cancel{\underline{\omega}} \times (\underline{\omega} \times \underline{OG}) \\ &= \begin{matrix} \times \\ \begin{pmatrix} -0.425, & 14.52, & 0 \end{pmatrix} \\ \begin{pmatrix} 0.6, & 0.2, & 0 \end{pmatrix} \end{matrix} \\ & \hline & \begin{pmatrix} 0, & 0, & -8.737 \end{pmatrix} \end{aligned}$$

Consequently, from (1)

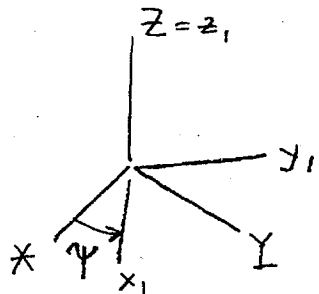
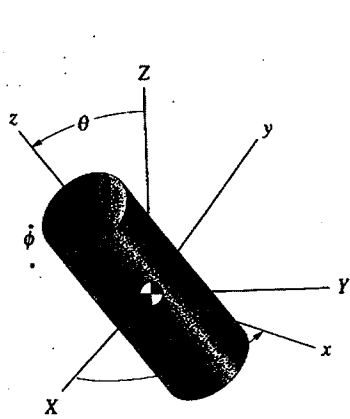
(5)

$$R_x = 0$$

$$R_y = 0$$

$$R_z = 49 + 5(-8.737) = \underline{\underline{5.33 \text{ [N]}}}$$

17. (bonus) A satellite can be modeled as an 1000-kg cylinder 4m in length and 2m in diameter. If the nutation angle is $\theta = 20^\circ$, and the spin rate $\dot{\phi}$ is one revolution per second, what is the satellite's precession rate $\dot{\psi}$ in revolutions per second? (25 points)



ang. velocity of x_1, y_1, z_1 wrt X, Y, Z

$$\underline{\omega}_1 = (0, 0, \dot{\psi})$$

(in both X, Y, Z, x_1, y_1, z_1)

ang. velocity of x, y, z wrt x_1, y_1, z_1 ,
 $\underline{\omega}_2 = \underline{\Omega}$ ($\theta = \text{const}$)

Transformation matrix from x_1, y_1, z_1 to x, y, z

$$\underline{\alpha} = \begin{pmatrix} 1 & 0 & 0 \\ 0 & \cos\theta & \sin\theta \\ 0 & -\sin\theta & \cos\theta \end{pmatrix}$$

$\underline{\omega}_1$ in system x, y, z

$$\begin{pmatrix} 1 & 0 & 0 \\ 0 & \cos\theta & \sin\theta \\ 0 & -\sin\theta & \cos\theta \end{pmatrix} \begin{pmatrix} 0 \\ 0 \\ \dot{\psi} \end{pmatrix} = \begin{pmatrix} 0 \\ \sin\theta \dot{\psi} \\ \cos\theta \dot{\psi} \end{pmatrix}$$

Calculations done in system x, y, z

$$\underline{\Omega} = \underline{\omega}_1 = (0, \sin\theta \dot{\psi}, \cos\theta \dot{\psi})$$

ang. velocity of the satellite wrt system x, y, z

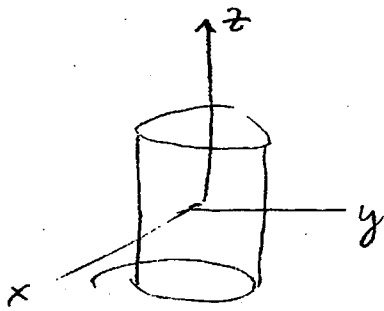
$$\underline{\omega}_3 = (0, 0, \dot{\phi}) \quad (\text{in } x, y, z)$$

ang. velocity of the satellite wrt X, Y, Z (in x, y, z !)

$$\underline{\omega} = \underline{\Omega} + \underline{\omega}_3 = (0, \sin\theta \dot{\psi}, \cos\theta \dot{\psi} + \dot{\phi})$$

(5)

(5)



$$0 < r < R$$

$$0 < \theta < 2\pi$$

$$-\frac{h}{2} < z < \frac{h}{2}$$

$$\begin{cases} x = r \cos \theta \\ y = r \sin \theta \\ z = z \end{cases}$$

$$I_x = \int_0^R \int_0^{2\pi} \int_{-\frac{h}{2}}^{\frac{h}{2}} \underbrace{(r^2 \sin^2 \theta + z^2)}_{y^2 + z^2} dz d\theta r dr$$

$$= \int \left(\frac{R^4}{4} h \cdot \pi + \frac{R^2}{2} \cdot \pi \frac{h^3}{12} \right) = \int \pi R^2 h \left(\frac{R^2}{4} + \frac{h^2}{12} \right)$$

$$\cos^2 \theta - \sin^2 \theta = \cos 2\theta$$

$$= \frac{1}{24} m (6R^2 + h^2)$$

$$1 - 2 \sin^2 \theta = \cos 2\theta$$

$$\sin^2 \theta = \frac{1 - \cos 2\theta}{2}$$

$$\int_0^{2\pi} \sin^2 \theta d\theta = \frac{1}{2} \left(\theta - \frac{1}{2} \sin 2\theta \right) \Big|_0^{2\pi} = \pi$$

$$I_z = \int_0^R \int_0^{2\pi} \int_{-\frac{h}{2}}^{\frac{h}{2}} (r^2 \sin^2 \theta + r^2 \cos^2 \theta) dz d\theta r dr$$

$$= \int \frac{R^4}{4} 2\pi h = \frac{1}{2} m R^2$$

$$I_z = \frac{1}{2} \cdot 800 \cdot 1^2 = 400 \text{ [kg m}^2\text{]}$$

$$I_x = \frac{1}{24} \cdot 800 (6 + 16) = 1267 \text{ [kg m}^2\text{]}$$

(5)

Equation of rotational motion

$$\dot{H}_c = M_c = 0$$

$$(\mathbf{I}_c \boldsymbol{\omega})^\cdot = 0$$

$$\cancel{\mathbf{I}_c \dot{\boldsymbol{\omega}}} + \boldsymbol{\Omega} \times (\mathbf{I}_c \boldsymbol{\omega}) = 0$$

(5)

Tensor of inertia at C

$$\mathbf{I}_c = \begin{pmatrix} I_x & 0 & 0 \\ 0 & I_x & 0 \\ 0 & 0 & I_z \end{pmatrix}$$

$$\times \boldsymbol{\Omega} \quad (0, \sin \theta \dot{\psi}, \cos \theta \dot{\psi})$$
$$\mathbf{I}_c \boldsymbol{\omega} = (0, I_x \sin \theta \dot{\psi}, I_z (\cos \theta \dot{\psi} + \dot{\varphi}))$$

$$(I_z (\cos \theta \dot{\psi} + \dot{\varphi}) \sin \theta \dot{\psi} - I_x \cos \theta \sin \theta \dot{\psi}^2, 0, 0)$$

$$(I_z - I_x) \cos \theta \sin \theta \dot{\psi}^2 + I_z \dot{\varphi} \sin \theta \dot{\psi} = 0$$

$$\dot{\psi} = \frac{I_z}{(I_x - I_z) \cos \theta} \dot{\varphi}$$

$$\dot{\psi} = \frac{500}{864 \cdot \cos 20^\circ} \cdot 1 = 0.49 \text{ [rad/s]}$$

(5)